CIVIL ENGINEERING DEPARTMENT

Introduction to Earthquake Engineering

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Outlines

- Introduction to earthquake engineering
- Introduction to earthquake seismology
- Dynamics of structures linear analysis
- Dynamics of structures nonlinear analysis
- Ductility and nonlinear behavior
- Design response spectra (UBC & IBC)
- Method of analysis Force based (equivalent lateral force)
- beam, column and joint design issues

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- Dynamics of <u>structures</u> ⇒ determination of response of structures under the effect of dynamic loading
- Dynamic load is one whose magnitude, direction, sense and point of application changes in time

Dynamics of structures

Dynamics of structures

p(t)















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Dynamics of structures

Single degree of freedom systems

- Simple structures:
 - stiffness k

Single degree of freedom systems (SDOF)

- Objective: find out response of SDOF system under the effect of:
 - a dynamic load acting on the mass
 - a seismic motion of the base of the structure
- The number of <u>degree of</u> <u>freedom (DOF)</u> necessary for dynamic analysis of a structure is the number of independent displacements necessary to define the displaced position of <u>masses</u> with respect to their initial position



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Single degree of freedom systems

- One-storey frame =
 - mass component
 - stiffness component
 - damping component
- Number of dynamic degrees of freedom = 1
- Number of static degrees of freedom = ?



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Force-displacement relationship



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Force-displacement relationship

- Linear elastic system:
 - elastic material
 - first order analysis

$$f_s = k \cdot u$$

 $f_{S} = f_{S}(u, \dot{u})$

- Inelastic system:
 - plastic material
 - First-order or second-order analysis

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Damping force

- <u>Damping</u>: decreasing with time of amplitude of vibrations of a system let to oscillate freely
- Cause: thermal effect of elastic cyclic deformations of the material and internal friction



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Damping

Damping in real structures:

- friction in steel connections
- opening and closing of microcracks in r.c. elements
- friction between structural and non-structural elements
- Mathematical description of these components impossible
- Modelling of damping in real structures ⇒ equivalent viscous damping

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 $f_D = c \cdot \dot{u}$

Damping

- Relationship between damping force and velocity:
 - c viscous damping coefficient units: (Force x Time / Length)
 - Determination of viscous damping:
 - free vibration tests
 - forced vibration tests
- Equivalent viscous damping ⇒ modelling of the energy dissipated by the structure in the elastic range



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SDOF systems: classical representation



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Equation of motion: seismic excitation

- Dynamics of structures in the case of seismic motion ⇒ determination of structural response under the effect of seismic motion applied at the base of the structure
- Ground displacement u_g
- Total (or absolute) displacement of the mass u^t
- Relative displacement between mass and ground u

 $u^{t}(t) = u(t) + u_{g}(t)$



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Dynamics of structures

Equation of motion: seismic excitation

- D'Alambert principle of dynamic equilibrium
- Elastic forces \Rightarrow relative displacement $u = f_s = k \cdot u$
- Damping forces \Rightarrow relative displacement $u = f_D = c \cdot \dot{u}$
- Inertia force \Rightarrow total displacement u^t $f_I = m\ddot{u}^t$



 $f_{I} + f_{s} + f_{D} = 0$ $m\ddot{u}^{t} + c\dot{u} + ku = 0$ $u^{t}(t) = u(t) + u_{g}(t)$ $m\ddot{u} + c\dot{u} + ku = -m\ddot{u}_{g}$

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Dynamics of structures

Equation of motion: seismic excitation

- Equation of motion in the case of an external force $m\ddot{u} + c\dot{u} + ku = p(t)$
- Equation of motion in the case of seismic excitation $m\ddot{u} + c\dot{u} + ku = -m\ddot{u}_{g}$
- Equation of motion for a system subjected to seismic motion described by ground acceleration ü_g is identical to that of a system subjected to an external force -mü_g
- Effective seismic force



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Dynamics of structures

Problem formulation

- Fundamental problem in dynamics of structures: determination of the response of a (SDOF) system under a dynamic excitation
 - a external force
 - ground acceleration applied to the base of the structure
 - "Response" ⇒ any quantity that characterizes behaviour of the structure
 - displacement
 - velocity
 - mass acceleration
 - forces and stresses in structural members

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Determination of element forces

 Solution of the equation of motion of the SDOF system ⇒ displacement time history u(t)

Displacements ⇒ forces in structural elements

- Imposed displacements \Rightarrow forces in structural elements
- <u>Equivalent static force</u>: an external static force *f_S* that produces displacements *u* determined from dynamic analysis

 $f_s(t) = ku(t)$

Forces in structural elements \Rightarrow by static analysis of the structure subjected to equivalent seismic forces f_S

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Dynamics of structures

Combination of static and dynamic response

- Linear elastic systems: superposition of effects possible ⇒ total response can be determined through the superposition of the results obtained from:
 - static analysis of the structure under permanent and live loads, temperature effects, etc.
 - dynamic response of the structure
- Inelastic systems: superposition of effects NOT possible ⇒ dynamic response must take account of deformations and forces existing in the structure before application of dynamic excitation





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Solution of the equation of motion

Equation of motion of a SDOF system

 $m\ddot{u}(t) + c\dot{u}(t) + ku(t) = p(t)$

differential linear non-homogeneous equation of second order

In order to completely define the problem:

- initial displacement u(0)
- initial velocity $\dot{u}(0)$

Solution methods:

- Classical solution
- Duhamel integral
- Numerical techniques

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Classical solution

- Complete solution u(t) of a linear non-homogeneous differential equation of second order is composed of
 - complementary solution u_c(t) and
 - particular solution $u_p(t)$
 - $u(t) = u_c(t) + u_p(t)$
- Second order equation ⇒ 2 integration constants ⇐ initial conditions
- Classical solution useful in the case of
 - free vibrations
 - forces vibrations, when dynamic excitation is defined analytically

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Classical solution: example

- Equation of motion of an undamped (*c*=0) SDOF system excited by a step force *p(t)=p₀*, *t≥0*:
 - $m\ddot{u} + ku = p_0$
- Particular solution: $u_p(t) = \frac{p_0}{k}$
- Complementary solution: $u_c(t) = A \cos \omega_n t + B \sin \omega_n t$
- where *A* and *B* are integration constants and $\omega_n = \sqrt{k/m}$
- The complete solution $u(t) = A \cos \omega_n t + B \sin \omega_n t + \frac{P_0}{k}$
- Initial conditions: for *t*=0 we have u(0) = 0 and $\dot{u}(0) = 0 \Rightarrow$

 $A = -\frac{p_0}{k}$ B = 0 the eq. of motion $u(t) = \frac{p_0}{k}(1 - \cos \omega_n t)$

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Undamped free vibrations

- General form of the equation of motion: $m\ddot{u} + c\dot{u} + ku = p(t)$
- Equation of motion in the case of undamped free vibrations:

 $m\ddot{u} + ku = 0$

- Vibrations ⇒ the system disturbed from the static equilibrium position by
 - initial displacement u(0)
 - initial velocity $\dot{u}(0)$
- Classical solution ⇒

 $u(t) = u(0) \cos \omega_n t + \frac{\dot{u}(0)}{\omega_n} \sin \omega_n t$ where $\omega_n = \sqrt{k/m}$

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Undamped free vibrations

Simple harmonic motion $u(t) = u(0)\cos\omega_n t + \frac{\dot{u}(0)}{\omega_n}\sin\omega_n t$



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Undamped free vibrations

- <u>Natural period of vibration</u> T_n time needed for an undamped SDOF system to perform a complete cycle of free vibrations
- Natural circular frequency

$$\int_{n} = \frac{2\pi}{\omega_n}$$

• <u>Natural frequency of vibration</u> f_n represents the number of complete cycles performed by the system in one second $f_n = \frac{1}{T}$ $f_n = \frac{\omega_n}{2\pi}$

$$\omega_n = \sqrt{k/m}$$

mass stiffness

 "Natural" - depends only on the properties of the SDOF system

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Undamped free vibrations

• Alternative expressions for ω_n , f_n , T_n :

$$\omega_n = \sqrt{\frac{g}{\delta_{st}}} \qquad f_n = \frac{1}{2\pi} \sqrt{\frac{g}{\delta_{st}}} \qquad T_n = 2\pi \sqrt{\frac{\delta_{st}}{g}}$$

 $\delta_{st} = mg/k$ elastic deformation of a SDOF system under a static force equal to mg

• Amplitude: magnitude of oscillations

$$u_0 = \sqrt{\left[u(0)\right]^2 + \left[\frac{\dot{u}(0)}{\omega_n}\right]^2}$$

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Damped free vibrations

- General form of the eq. of motion: $m\ddot{u} + c\dot{u} + ku = p(t)$
- Equation of motion in the case of damped free vibrations: $m\ddot{u} + c\dot{u} + ku = 0$
- Dividing eq. by m we obtain

$$\ddot{u} + 2\xi\omega_n\dot{u} + \omega_n^2u = 0$$

- with the notations: $\omega_n = \sqrt{k/m}$ $\xi = \frac{c}{2m\omega} = \frac{c}{c}$
- Critical damping coefficient

$$c_{cr} = 2m\omega_n = 2\sqrt{km} = \frac{2k}{\omega_n}$$

- Damping coefficient c a measure of the energy dissipated in a complete cycle
- ξ critical damping ratio: a non-dimensional measure of damping, which depends on the stiffness and mass as well

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Types of motion

- c=c_{cr} or ξ = 1 ⇒ the system returns to the position of equilibrium without oscillation
- $c > c_{cr}$ or $\xi > 1 \Rightarrow$ the system returns to the position of equilibrium without oscillation, but slower
- $c < c_{cr}$ or $\xi < 1 \Rightarrow$ the system oscillates with respect to the equilibrium position with progressively decreasing amplitudes



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Types of motion

- c_{cr} the smallest value of the damping coefficient that completely prevents oscillations
- Most engineering structures underdamped (c<c_{cr})
- Few reasons to study:
 - critically damped systems (*c=c_{cr}*)
 - overdamped systems (c>c_{cr})

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Damping for engineering structures

stress level	structural type	ξ (%)
stress level below 0.5 times the yield strength	welded steel structures, prestressed concrete structures, strongly reinforced concrete structures (limited cracks)	2-3
	reinforced concrete structures with significant cracking	3-5
	steel structures with bolted or riveted connections, wood structures connected with screws or nails	5-7
stresses close to the yield strength	welded steel structures, prestressed concrete structures (without total loss of prestress)	5-7
	prestressed concrete structures with total loss of prestress	7-10
	reinforced concrete structures	7-10
	steel structures with bolted or riveted connections, wood structures connected with screws	10-15
	wood structures connected with nails	15-20



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Dynamics of structures

Seismic action

North-south component of the El Centro, California record during Imperial Valley earthquake from 18.05.1940 0.450 üę.) $\ddot{u}_{go} = 0.319g$ -0.4 15 úg, in./sec O $\dot{u}_{go} = 13.04 \text{ in./sec}$ -15 *u_g*, in. $u_{go} = 8.40$ in. -10 5 10 0 15 2025 30 Time, sec

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Determination of seismic response

Equation of motion: $m\ddot{u} + c\dot{u} + ku = -m\ddot{u}_{g}$

/m:
$$\ddot{u} + 2\xi \omega_n \dot{u} + \omega_n^2 u = -\ddot{u}_g$$

- Numerical methods
 - central difference method
 - Newmark method

...

$$\bigg\} \Rightarrow u \equiv u(t, T_n, \xi)$$

- Response depends on:
 - natural circular frequency ω_n (or natural period T_n)
 - critical damping ratio ξ
 - ground motion \ddot{u}_{g}

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Seismic response $T_n = 2 \sec, \zeta = 0$ $T_n = 0.5 \text{ sec}, \zeta = 0.02$ 9.91 in. 10 -0 2.67 in. -10- $T_n = 2 \sec, \zeta = 0.02$ $T_n = 1 \text{ sec}, \zeta = 0.02$ ר 10 0 5.97 in. 7.47 in. -10 $T_n = 2 \sec, \zeta = 0.02$ $T_n = 2 \sec, \zeta = 0.05$ 10 1 5.37 in. , 7.47 in. -10 30 30 20 20 10 0 10 0 Time, sec Time, sec

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Elastic response spectra

 Response spectrum: representation of peak values of seismic response (displacement, velocity, acceleration) of a SDOF system versus natural period of vibration, for a given critical damping ratio

$$u_0(T_n,\xi) = \max_t |u(t,T_n,\xi)|$$
$$\dot{u}_0(T_n,\xi) = \max_t |\dot{u}(t,T_n,\xi)|$$
$$\ddot{u}_0^t(T_n,\xi) = \max_t |\ddot{u}^t(t,T_n,\xi)|$$



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Pseudo-velocity and pseudo-acceleration

Spectral pseudo-velocity: $V = \omega_n D = \frac{2\pi}{T} D$ units of velocity different from peak velocity $E_{s0} = \frac{ku_0^2}{2} = \frac{kD^2}{2} = \frac{k\left(V/\omega_n\right)^2}{2} = \frac{mV^2}{2}$

Strain energy

Spectral pseudo-acceleration:

$$f_{s0} = ku_0 = m\omega_n^2 u_0 = mA \implies A = \omega_n^2 u_0 = \omega_n^2 D = \left(\frac{2\pi}{T_n}\right) D$$

>2

- units of acceleration
- different from peak acceleration

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Combined D-V-A spectrum

- Displacement, pseudo-velocity and pseudo-acceleration spectra:
 - same information
 - different physical meaning

$$\frac{A}{\omega_n} = V = \omega_n D \quad or \quad \frac{T_n}{2\pi} A = V = \frac{2\pi}{T_n} D$$

 $(T_n/2\pi)A = V \implies \lg T_n + \lg A - \lg 2\pi = \lg V$

- A line inclined at +45[◦] for IgA Ig2π = const. ⇒ spectral pseudo-acceleration: an axis inclined to -45[◦]
- Similarly, spectral displacement: an axis inclined to +45^o

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Combined D-V-A spectrum



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Characteristics of elastic response spectra



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Characteristics of elastic response spectra

- For $T_n < T_a$
 - pseudo-acceleration A is close to $\ddot{\mathcal{U}}_{g0}$
 - spectral displacement D is small
- For $T_n > T_f$
 - spectral displacement **D** is close to u_{g0}
 - spectral pseudo-acceleration A is small
- Between T_a and $T_c \Rightarrow A > \ddot{u}_{g0}$
- Between T_b and $T_c \Rightarrow A$ can be considered constant
- Between T_d and $T_f \Rightarrow D > u_{g0}$
- Between T_d and $T_e \Rightarrow D$ can be considered constant
- Between T_c and $T_d \Rightarrow V > \dot{u}_{g0}$
- Between T_c and $T_d \Rightarrow V$ can be considered constant



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Characteristics of elastic response spectra

- $T_n > T_d \Rightarrow$ response region sensible to displacements
- *T_n*<*T_c* ⇒ response region sensible to accelerations
- $T_c < T_n < T_d \Rightarrow$ response region sensible to velocity

Larger damping:

- smaller values of displacements, pseudo-velocity and pseudoacceleration
- more "smooth" spectra
- Effect of damping:
 - insignificant for $T_n \rightarrow 0$ and $T_n \rightarrow \infty$,
 - important for $T_b < T_n < T_d$

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Elastic design spectra

- Spectra of past ground motions:
 - jagged shape
 - variation of response for different earthquakes
 - areas where previous data is not available



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Elastic design spectra

- idealized "smooth" spectra
- based on statistical interpretation (median; median plus standard deviation)





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Elastic design spectra



Natural vibration period (log scale)

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Dynamics of structures

Inelastic response of SDOF systems

- Most structures designed for seismic forces lower than the ones assuring an elastic response during the design earthquake
 - design of structures in the elastic range for rare seismic events considered uneconomical
 - in the past, structures designed for a fraction of the forces necessary for an elastic response, survived major earthquakes



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Inelastic response of SDOF systems

- Elasto-plastic system:
 - stiffness k
 - yield force f_y
 - yield displacement u_y
- Elasto-plastic idealization: equal area under the actual and idealised curves up to the maximum displacement u_m

 Cyclic response of the elastoplastic system



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Dynamics of structures

Corresponding elastic system

- Corresponding elastic system:
 - same stiffness
 - same mass
 - same damping
- the same period of vibration (at
 - small def.)
- Inelastic response:
 - yield force reduction factor R_y



- ductility factor

$$\mu = \frac{u_m}{u_y}$$



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Dynamics of structures

Equation of motion

• Equation of motion: $m\ddot{u} + c\dot{u} + f_s(u, \dot{u}) = -m\ddot{u}_g$

$$/\mathbf{m} \implies \ddot{u} + 2\xi \omega_n \dot{u} + \omega_n^2 u_y \tilde{f}_s (u, \dot{u}) = -\ddot{u}_g$$
$$\tilde{f}_s (u, \dot{u}) = f_s (u, \dot{u}) / f_y$$

- Seismic response of an inelastic SDOF system depends on:
 - natural circular frequency of vibration ω_n
 - critical damping ratio ξ
 - yield displacement u_y
 - force-displacement shape $f_s(u, \dot{u})$

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Dynamics of structures

Effects of inelastic force-displacement relationship

- 4 SDOF systems (El Centro):
 - $T_n = 0.5 \, \text{sec}$
 - ξ**= 5%**
 - R_y = 1, 2, 4, 8
- Elastic system:
 - vibr. about the initial position of equilibrium

Deplasare u, in

- u_p=0

- Inelastic syst.:
 vibr. about a
 - vibr. about a new position of equilibrium

– *u_p*≠0





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Dynamics of structures

Elastic ⇔ inelastic

- Design of a structure responding in the elastic range:
 f₀ ≤ f_{Rd}
- Design of a structure responding in the inelastic range: *u_m* ≤ *u_{Rd}* μ ≤ μ_{Rd}

ductility demand ductility capacity



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Dynamics of structures

 u_m/u_0 ratio



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Dynamics of structures

R_y - μ relationship: idealisation

- T_n in the displacement- and velocity-sensitive region:
 - "equal displacement" rule $u_m/u_0=1 \Rightarrow R_y=\mu$
- *T_n* in the acceleration-sensitive region:
 - "equal energy" rule $u_m/u_0 > 1 \Rightarrow R_y = \sqrt{2\mu 1}$
- *T_n<T_a:*

- small deformations, elastic response $\Rightarrow R_v = 1$



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R_y - μ relationship: idealisation



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Dynamics of structures

Multi degree of freedom systems

- Structure idealisation: elements interconnected at nodes
- Degrees of freedom: node displacements/rotations
 - 3 DOF for two-dimensional frames
 - 6 DOF for three-dimensional frames







Forces applied at nodes

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Dynamics of structures

Elastic forces

- Node displacements $u_i \Leftrightarrow$ nodal forces f_{Si}
- Linear systems: nodal forces determined based on
 - superposition principle
 - stiffness coefficients
- Stiffness coefficient k_{ij} is equal to the force along DOF i due to a unit displacement along DOF j



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Dynamics of structures

Elastic forces

 Knowing stiffness coefficients k_{ij}, nodal forces f_{Si} along DOF i, associated with displacements u_j, j=1, 2, ..., N are obtained by superposition

 $f_{Si} = k_{i1}u_1 + k_{i2}u_2 + \dots + k_{ij}u_j + \dots + k_{iN}u_N$

Equations corresponding to *i*=1, 2, ..., N can be written in matrix form

$$\begin{bmatrix} f_{S1} \\ f_{S2} \\ \vdots \\ f_{SN} \end{bmatrix} = \begin{bmatrix} k_{11} & k_{12} & \dots & k_{1j} & \dots & k_{1N} \\ k_{21} & k_{22} & \dots & k_{2j} & \dots & k_{2N} \\ \vdots & \vdots & & \vdots & & \vdots \\ k_{N1} & k_{N2} & \dots & k_{Nj} & \dots & k_{NN} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_N \end{bmatrix}$$

 $[f_s] = [k] \{u\}$

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Dynamics of structures

Damping forces

- A unit velocity along DOF j, generates forces along considered DOFs
- Damping coefficient c_{ij} is the force along DOF i due to a unit velocity along DOF j



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Dynamics of structures

Damping forces

 Knowing damping coefficients c_{ij}, nodal forces f_{Di} along DOF *i*, associated to velocity *u*_j *j*=1, 2, ..., N are obtained by superposition

$$f_{Di} = c_{i1}\dot{u}_1 + c_{i2}\dot{u}_2 + \dots + c_{ij}\dot{u}_j + \dots + c_{iN}\dot{u}_N$$

 Equations corresponding to *i*=1, 2, ..., N can be written in matrix form

$$\begin{bmatrix} f_{D1} \\ f_{D2} \\ \vdots \\ f_{DN} \end{bmatrix} = \begin{bmatrix} c_{11} & c_{12} & \dots & c_{1j} & \dots & c_{1N} \\ c_{21} & c_{22} & \dots & c_{2j} & \dots & c_{2N} \\ \vdots & \vdots & & \vdots & & \vdots \\ c_{N1} & c_{N2} & \dots & c_{Nj} & \dots & c_{NN} \end{bmatrix} \begin{bmatrix} \dot{u}_1 \\ \dot{u}_2 \\ \vdots \\ \dot{u}_k \end{bmatrix}$$
$$\begin{bmatrix} f_D \end{bmatrix} = \begin{bmatrix} c \end{bmatrix} \{ \dot{u} \}$$

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Dynamics of structures

Inertia forces

 A unit acceleration along DOF j, according to D'Alambert principle will generate fictitious inertia forces along the considered DOFs

The mass coefficient *m_{ij}* is the force along DOF *i* due to a unit acceleration along DOF *j*



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Dynamics of structures

Inertia forces

 Knowing the mass coefficients m_{ij}, nodal forces f_{ii} along DOF *i*, associated to acceleration ü_j *j*=1, 2, ..., N are obtained by superposition

 $f_{Ii} = m_{i1}\ddot{u}_1 + m_{i2}\ddot{u}_2 + \ldots + m_{ij}\ddot{u}_j + \ldots + m_{iN}\ddot{u}_N$

 Equations corresponding to *i*=1, 2, ..., N can be written in matrix form

$$\begin{bmatrix} f_{I1} \\ f_{I2} \\ \vdots \\ f_{IN} \end{bmatrix} = \begin{bmatrix} m_{11} & m_{12} & \dots & m_{1j} & \cdots & m_{1N} \\ m_{21} & m_{22} & \cdots & m_{2j} & \cdots & m_{2N} \\ \vdots & \vdots & & \vdots & & \vdots \\ m_{N1} & m_{N2} & \cdots & m_{Nj} & \cdots & m_{NN} \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \\ \vdots \\ \ddot{u}_N \end{bmatrix}$$
$$\begin{bmatrix} f_I \end{bmatrix} = \begin{bmatrix} m \end{bmatrix} \{ \ddot{u} \}$$

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Dynamics of structures

Mass idealisation: 2D structures

- Generally mass is distributed through the structure
- In a simplified manner ⇒ concentrated in nodes
- Rotational component: generally neglected
- Neglecting axial deformations of members ⇒ masses can be considered concentrated at the floor levels
- In general, for masses lumped in nodes, the mass matrix is diagonal





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Dynamics of structures

Mass idealisation: 3D structures

- Multistorey three-dimensional structures: number of elements in the mass matrix can be reduced due to floor diaphragm effect
 - infinite in-plane stiffness
 - flexible out of plane
- Rigid floor diaphragms ⇒ 3 DOFs defined at the center of mass: u_x, u_y, u_θ
- Flexible floor diaphragms ⇒ masses should be assigned to each node, proportionally to their tributary area





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Dynamics of structures

Equation of motion: dynamic force

- Dynamic forces {p(t)} can be considered distributed to:
 - stiffness component $\{f_s(t)\}$
 - damping component $\{f_D(t)\}$
 - mass component
 - $\{f_{I}(t)\}+\{f_{D}(t)\}+\{f_{S}(t)\}=\{p(t)\}$
- Equation of motion: $[m]{\ddot{u}} + [c]{\dot{u}} + [k]{u} = {p(t)}$

 $\{f_I(t)\}$



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Dynamics of structures

Equation of motion: ground motion

- MDOF systems with al DOFs displacements in the same direction with ground motion
 - ground displacement: u_g
 - total displacement of mass m_j: u^t_j
 - relative displacement between mass and ground: u_i

 $\left\{u_{j}^{t}\left(t\right)\right\} = \left\{u_{j}\left(t\right)\right\} + u_{g}\left(t\right)\left\{1\right\}$



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Dynamics of structures

Equation of motion: ground motion

In case of ground motion dynamic forces $\{p(t)\}=0$ $\{f_{I}(t)\}+\{f_{D}(t)\}+\{f_{S}(t)\}=\{0\}$ $[m]\{\ddot{u}^{t}\}+[c]\{\dot{u}\}+[k]\{u\}=\{0\}$ $\{u_{j}^{t}(t)\}=\{u_{j}(t)\}+u_{g}(t)\{1\}$ $[m]\{\ddot{u}\}+[c]\{\dot{u}\}+[k]\{u\}=-[m]\{1\}\ddot{u}_{g}(t)$





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Dynamics of structures

Free vibrations of MDOF systems

- Equation of motion :[m]{ü}+[k]{u} = {0}
 I
 a system of N homogeneous differential equations
- Initial conditions: $\{u\} = \{u(0)\}$ $\{\dot{u}\} = \{\dot{u}(0)\}$



- The motion is NOT harmonic
- Deformed shape of the structure changes in time

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Dynamics of structures

Free vibrations of MDOF systems

- For an appropriate distribution of initial deformations:
 {u} = {u(0)}
- Vibrations ARE harmonic
- Deformed shape does NOT change in time
 - 1st Distribution \Box 1st natural mode $\{\phi\}_1$

1st natural period T₁

U2

 u_1

m

2m

7777

k

2k

7777



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Dynamics of structures

Free vibrations of MDOF systems

- For an appropriate distribution of initial deformations:
 {u} = {u(0)}
- Vibrations ARE harmonic
- Deformed shape does NOT change in time
- 2nd Distribution ⇒

2nd natural mode $\{\phi\}_2$

2nd natural period T₂





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Dynamics of structures

Natural modes of undamped MDOF systems

- Vibrations in n-th natural mode: {u(t)}_n = q_n(t){Ø}_n
 - deformed shape: {\u03c6} }
 - time response: $q_n(t) = A_n \cos \omega_n t + B_n \sin \omega_n t$

$$\left\{u(t)\right\}_{n} = \left\{\phi\right\}_{n} \left(A_{n} \cos \omega_{n} t + B_{n} \sin \omega_{n} t\right) \qquad [m]\left\{\ddot{u}\right\} + [k]\left\{u\right\} = \left\{0\right\}$$

$$\left[-\omega_n^2[m]\{\phi\}_n + [k]\{\phi\}_n\right]q_n(t) = \{0\}$$

q_n(t)=0 trivial solution $[k]\{\phi\}_{n} = \omega_{n}^{2}[m]\{\phi\}_{n} \text{ or } ([k] - \omega_{n}^{2}[m])\{\phi\}_{n} = \{0\}$ non-trivial solution eigenvalue problem \bigcup determination of scalars ω_{n} and vectors $\{\phi\}_{n}$ $\det([k] - \omega_{n}^{2}[m]) = 0$

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Dynamics of structures

Natural modes of undamped MDOF systems

- **Expanding the determinant** $det([k] \omega_n^2[m]) = 0$
- Characteristic equation: polynomial of order N in ω_n^2
 - N eigenvalues ω_n^2
 - *N* eigenmodes $\{\phi\}_n$ (relative values just shape)
 - Matrix notation: - eigenmodes: $[\Phi] = \{\{\phi\}_1 \cdots \{\phi\}_n\} = \begin{bmatrix} \phi_{11} \cdots \phi_{1N} \\ \vdots & \ddots & \vdots \\ \phi_{N1} \cdots & \phi_{NN} \end{bmatrix}$ - eigenvalues: $[\Omega^2] = \begin{bmatrix} \omega_1^2 & & \\ & \ddots & \\ & & & \omega_N^2 \end{bmatrix}$

- eigenproblem: $[k][\Phi] = [m][\Phi][\Omega^2]$
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Dynamics of structures

Orthogonality of natural modes

- **Eigenproblem:** $|k| \{\phi\}_{n} = \omega_{n}^{2} [m] \{\phi\}_{n}$
- Multiplying to the left by $\{\phi\}_{r}^{T}(r\neq n)$:

transpose $\{\phi\}_{r}^{T}[k]\{\phi\}_{n} = \omega_{n}^{2}\{\phi\}_{r}^{T}[m]\{\phi\}_{n} \quad \Longrightarrow \quad \{\phi\}_{n}^{T}[k]\{\phi\}_{r} = \omega_{n}^{2}\{\phi\}_{n}^{T}[m]\{\phi\}_{r} \quad \textbf{(4.33)}$

- $\{\phi\}_{n}^{T}[k]\{\phi\}_{r} = \omega_{r}^{2}\{\phi\}_{n}^{T}[m]\{\phi\}_{r}$ (4.32) Similarly:
- **Difference between (4.33) and (4.32):** $(\omega_n^2 \omega_r^2) \{\phi\}_n^T [m] \{\phi\}_r = 0$
- For $\omega_n^2 \neq \omega_r^2$, which for positive values implies $\omega_n \neq \omega_r$: (4.32) $\{\phi\}_{n}^{T}[m]\{\phi\}_{r}=0$ \implies $\{\phi\}_{n}^{T}[k]\{\phi\}_{r}=0$ orthogonality of natural modes
 - **Matrices M and K:** $[K] \equiv [\Phi]^T [k] [\Phi]$ $[M] \equiv [\Phi]^T [m] [\Phi]$ diagonal

 $K_{n} = \{\phi\}_{n}^{T} [k] \{\phi\}_{n} \qquad M_{n} = \{\phi\}_{n}^{T} [m] \{\phi\}_{n} \quad K_{n} = \omega_{n}^{2} M_{n}$

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Dynamics of structures

Normalisation of modes

- Natural modes: vectors for which only relative values are known
- Normalisation of modes:
 - setting the maximum value of a natural mode to unity
 - setting the value corresponding to a characteristic DOF to unity
 - normalisation of natural modes so that M_n are unity (normalisation with respect to the [m] matrix)

 $M_{n} = \{\phi\}_{n}^{T} [m] \{\phi\}_{n} = 1 \qquad [\Phi]^{T} [m] [\Phi] = [I]$ $K_{n} = \{\phi\}_{n}^{T} [k] \{\phi\}_{n} = \omega_{n}^{2} M_{n} = \omega_{n}^{2} \qquad [K] = [\Phi]^{T} [k] [\Phi] = [\Omega^{2}]$

orthonormal natural modes

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Dynamics of structures

Modal expansion of displacements

 Any set of N independent vectors can be used to express another vector of order N

 $\{u\} = \sum_{r=1}^{n} \{\phi\}_r q_r = [\Phi]\{q\}$ (4.48)

q_r - modal coordinates

• Multiplying both sides of (4.48) by $\{\phi\}_n^T[m]$ $\{\phi\}_n^T[m]\{u\} = \sum_{r=1}^N \{\phi\}_n^T[m]\{\phi\}_r q_r$

all terms are equal to zero, excepting those corresponding to r=n: $\{\phi\}_{n}^{T}[m]\{u\} = \{\phi\}_{n}^{T}[m]\{\phi\}_{n}q_{n}$

• Modal coordinates can be determined: $q_n = \frac{\{\phi\}_n^T [m]\{u\}}{\{\phi\}_n^T [m]\{\phi\}_n} = \frac{\{\phi\}_n^T [m]\{u\}}{M_n}$

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Dynamics of structures

Solution of equation of motion

• Equation of motion $[m]{\ddot{u}} + [k]{u} = {0}$

initial conditions

 $\{u\} = \{u(0)\}$ $\{\dot{u}\} = \{\dot{u}(0)\}$

• Eigenproblem:

$$([k] - \omega_n^2[m]) \{ \phi \}_n = \{ 0 \} \implies \boldsymbol{\omega}_n , \phi_n$$

- **Response in mode** *n*: $\{u(t)\}_n = \{\phi\}_n (A_n \cos \omega_n t + B_n \sin \omega_n t)$
- General solution ⇒ superposition of individual response in each natural mode:
 - $\left\{u(t)\right\} = \sum_{n=1}^{\infty} \left\{\phi\right\}_{n} \left(A_{n} \cos \omega_{n} t + B_{n} \sin \omega_{n} t\right)$ (4.52)
- Velocity vector:

$$\left\{\dot{u}(t)\right\} = \sum_{n=1}^{N} \left\{\phi\right\}_{n} \omega_{n} \left(-A_{n} \sin \omega_{n} t + B_{n} \cos \omega_{n} t\right)$$

• For *t*=0: $\{u(0)\} = \sum_{n=1}^{N} \{\phi\}_n A_n$ $\{\dot{u}(0)\} = \sum_{n=1}^{N} \{\phi\}_n \omega_n B_n$ (4.54) a system of *N* linear algebraic equations with unknowns A_n , respectively B_n

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Dynamics of structures

Solution of equation of motion

- Using modal expansion, vectors {u(0)} and {u(0)} can be written as
 - $\left\{u(0)\right\} = \sum_{n=1}^{N} \left\{\phi\right\}_{n} q_{n}(0) \qquad \left\{\dot{u}(0)\right\} = \sum_{n=1}^{N} \left\{\phi\right\}_{n} \dot{q}_{n}(0) \qquad (4.55)$

where modal coordinates are given by: $q_n(0) = \frac{\{\phi\}_n^T[m]\{u(0)\}}{M_n} \qquad \dot{q}_n(0) = \frac{\{\phi\}_n^T[m]\{\dot{u}(0)\}}{M_n}$

Equations (4.54) and (4.55) are equivalent

$$\{u(0)\} = \sum_{n=1}^{N} \{\phi\}_{n} A_{n} \qquad \{\dot{u}(0)\} = \sum_{n=1}^{N} \{\phi\}_{n} \omega_{n} B_{n} \quad (4.54) \Longrightarrow$$

$$A_{n} = q_{n}(0) \qquad B_{n} = \dot{q}_{n}(0) / \omega_{n}$$

Replacing these expressions in (4.52) ⇒

$$\left\{u(t)\right\} = \sum_{n=1}^{N} \left\{\phi\right\}_{n} \left(q_{n}(0)\cos\omega_{n}t + \frac{\dot{q}_{n}(0)}{\omega_{n}}\sin\omega_{n}t\right)$$

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Dynamics of structures

Free vibrations of MDOF systems with damping

- Equation of motion $[m]{\ddot{u}} + [c]{\dot{u}} + [k]{u} = \{0\}$
 - Initial conditions: $\{u\} = \{u(0)\}$ $\{\dot{u}\} = \{\dot{u}(0)\}$
 - Using modal expansion, equation of motion becomes: [m][Φ]{ÿ}+[c][Φ]{ý}+[k][Φ]{q}={0}

$$\{u\} = \sum_{r=1}^{N} \{\phi\}_{r} q_{r} = [\Phi]\{q\}$$

- Multiplying to the left by $[\Phi]^T$: $[M]{\ddot{q}}+[C]{\dot{q}}+[K]{q}={0}$ where: $[K] \equiv [\Phi]^T [k][\Phi] \qquad [M] \equiv [\Phi]^T [m][\Phi] \qquad [C] = [\Phi]^T [c][\Phi]$
- For a diagonal matrix [C] classical damping:
 - natural modes of the damped system identical to the ones of the undamped system
 - most engineering structures can be modelled using classical damping

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Dynamics of structures

Free vibrations of MDOF systems with damping

- Natural modes of the damped system identical to those of the undamped system - {\u03c6}_n
- Displacements similar to those of the undamped system, but amplitudes decrease with time
- Response of each mass is harmonic, similarly to that of a SDOF system



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Dynamics of structures

Free vibrations of MDOF systems with damping

 For each natural mode n, equation of motion in modal coordinates is

 $M_n \ddot{q}_n + C_n \dot{q}_n + K q_n = 0$

• Dividing by M_n one gets: $\ddot{q}_n + 2\xi_n \omega_n \dot{q}_n + \omega_n^2 q_n = 0$

where
$$\xi_n = \frac{C_n}{2M_n \omega_n}$$

 The same form as the equation of motion in the case of damped free vibrations of SDOF systems ⇒

$$q_n(t) = e^{-\xi_n \omega_n t} \left[q_n(0) \cos \omega_{nD} t + \frac{\dot{q}(0) + \xi_n \omega_n q_n(0)}{\omega_{nD}} \sin \omega_{nD} t \right] \qquad \qquad \omega_{nD} = \omega_n \sqrt{1 - \xi_n^2}$$

Combining modal contributions:

$$\left\{u\left(t\right)\right\} = \sum_{n=1}^{N} \left\{\phi\right\}_{n} e^{-\xi_{n}\omega_{n}t} \left[q_{n}(0)\cos\omega_{nD}t + \frac{\dot{q}(0) + \xi_{n}\omega_{n}q_{n}(0)}{\omega_{nD}}\sin\omega_{nD}t\right]$$

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Dynamics of structures

Modal analysis

 Equation of motions of a MDOF system with damping excited by dynamic forces:

 $[m]{\ddot{u}}+[c]{\dot{u}}+[k]{u}={p(t)}$ 4.82

- Displacements $\{u\}$ can be expanded as: $\{u\} = \sum_{r=1}^{N} \{\phi\}_r q_r = [\Phi]\{q\}$
- Replacing {u} in 4.82:

 $\sum_{r=1}^{N} [m] \{\phi\}_{r} \ddot{q}_{r}(t) + \sum_{r=1}^{N} [c] \{\phi\}_{r} \dot{q}_{r}(t) + \sum_{r=1}^{N} [k] \{\phi\}_{r} q_{r}(t) = \{p(t)\}$ **4.84**

• Multiplying 4.84 to the left by $\{\phi\}_n^T$ we obtain:

 $\sum_{r=1}^{N} \{\phi\}_{n}^{T} [m] \{\phi\}_{r} \ddot{q}_{r}(t) + \sum_{r=1}^{N} \{\phi\}_{n}^{T} [c] \{\phi\}_{r} \dot{q}_{r}(t) + \sum_{r=1}^{N} \{\phi\}_{n}^{T} [k] \{\phi\}_{r} q_{r}(t) = \{\phi\}_{n}^{T} \{p(t)\}$

• Which, considering orthogonality of natural modes becomes: $M_n \ddot{q}_n(t) + C_n \dot{q}_n(t) + K_n q_n(t) = P_n(t)$ 4.86

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Modal analysis

- Dividing by M_n one gets: $\ddot{q}_n + 2\xi_n \omega_n \dot{q}_n + \omega_n^2 q_n = \frac{P_n(t)}{M_n}$
- Solving <u>a system</u> of N differential equations was reduced to solution of <u>N independent equations</u>
- Direct estimation of the damping matrix [c] not necessary
- The same form with the equation of motions of a SDOF system ⇒ same solution methods
- Solution: modal coordinate q_n(t) for mode n
- Contribution of mode *n* to total displacement $\{u(t)\}_{n} = \{\phi\}_{n} q_{n}(t)$
 - Total displacements:

$$\{u(t)\} = \sum_{n=1}^{N} \{u(t)\}_{n} = \sum_{n=1}^{N} \{\phi\}_{n} q_{n}(t)$$

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Modal analysis

- Analysis procedure is called <u>modal analysis</u> and is applicable only to linear systems with classical damping
- Element forces can be obtained using 2 methods:
- Contributions r_n(t) in n-th mode are obtained from imposing displacements {u(t)}_n Total forces are obtained by superposition of modal contributions

$$r(t) = \sum_{n=1}^{N} r_n(t)$$

2. Equivalent static forces from the *n*-th mode are determined:

$$\{f(t)\}_{n} = [k]\{u(t)\}_{n} = \omega_{n}^{2}[m]\{u(t)\}_{n} = \omega_{n}^{2}[m]\{\phi\}_{n} q_{n}(t)$$

Static analysis \Rightarrow modal contributions $r(t) = \sum_{n=1}^{N} r_n(t)$ $r_n(t)$ from the *n*-th mode \Rightarrow

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Modal analysis: summary

- Define the structural properties
 - mass [m] and stiffness [k] matrices
 - critical damping ratio ξ_n
- Determine natural circular frequencies ω_n and natural modes of vibrations $\{\phi\}_n$
- Compute response in each mode following the sequence:
 - set up equation of motion $\ddot{q}_n + 2\xi_n \omega_n \dot{q}_n + \omega_n^2 q_n = \frac{P_n(t)}{M}$
 - compute modal displacements {u(t)}
 - compute element forces $r_n(t)$ from the *n*-th mode
- Combine modal contributions to obtain the total response
- $\left\{u\left(t\right)\right\} = \sum_{n=1}^{n} \left\{u\left(t\right)\right\}_{n}$
- Note: generally it is NOT necessary to consider ALL modes of vibration

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Modal analysis of seismic response: summary

- Define numerically ground acceleration
 - Define the structural properties
 - mass [m] and stiffness [k] matrices
 - critical damping ratio ξ_n
- Determine \omega_n and \{\phi\}_n
- Determine modal components {s}_n of the distribution of effective seismic forces
- Compute response in each mode following the sequence:
 - static response r_n^{st} of the structure from $\{s\}_n$
 - pseudo-acceleration A_n(t) of n-th mode SDOF system
 - resp. quantities $r_n(t)$ from the n-th mode $r_n(t) = r_n^{st} A_n(t)$
- Combine modal contributions to obtain the total response

$$r(t) = \sum_{n=1}^{N} r_n(t) = \sum_{n=1}^{N} r_n^{st} A_n(t)$$

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Effective modal mass

 Modal analysis - equivalent static forces in *n*-th mode ⇒ *n*-th mode contributions *r_n(t)* to the response quantity *r(t)*:

 $\left\{ f\left(t\right) \right\}_{n} = \left\{ s \right\}_{n} A_{n}\left(t\right)$ $\left\{ s \right\}_{n} = \Gamma_{n} \left[m\right] \left\{\phi\right\}_{n} \qquad s_{jn} = \Gamma_{n} m_{j} \phi_{jn}$

$$\Gamma_n = \frac{L_n}{M_n}$$

$$L_n = \{\phi\}_n^T [m]\{1\} = \sum_{j=1}^N m_j \phi_{jn}$$

$$M_n = \{\phi\}_n^T [m]\{\phi\}_n = \sum_{j=1}^N m_j \phi_{jn}^2$$

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Effective modal mass

 Response quantity r_n(t) can be expressed by: r_n(t) = rst_nA_n(t)



$$\{s\}_n = \Gamma_n[m]\{\phi\}_n \qquad s_{jn} = \Gamma_n m_j \phi_{jn}$$

 Multistorey structures: base shear force V_b

$$V_{bn}^{st} = \sum_{j=1}^{N} s_{jn} = \Gamma_n \sum_{j=1}^{n} m_j \phi_{jn} = \Gamma_n L_n \equiv M_n^*$$
$$M_n^* = \Gamma_n L_n = \left(\sum_{j=1}^{n} m_j \phi_{jn}\right)^2 / \sum_{j=1}^{n} m_j \phi_{jn}^2$$



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Effective modal mass

Base shear force in *n*-th mode:

 $V_{bn}(t) = V_{bn}^{st} A_n(t)$

substituting $V_{bn}^{st} = M_n^* \implies V_{bn}(t) = M_n^* A_n(t)$ 4.113

• A SDOF system with mass *m*, natural circular frequency ω_n and critical damping ratio ξ_n

 $V_b(t) = mA_n(t) \qquad 4.114$

- Comparing 4.113 and 4.114 $\Rightarrow M_n^*$ effective modal mass
- MDOF: only the portion M_n^{*} of the total mass of the structure is effective in producing the base shear force
- The sum of effective modal masses over all N modes is equal to the total mass of the structure N N

$$\sum_{n=1}^{N} M_{n}^{*} = \sum_{j=1}^{N} m_{j}$$

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Spectral analysis

- Modal analysis: time-history response $r_n(t) = r_n^{st} A_n(t)$ $\square r(t) = \sum_{n=1}^N r_n(t)$ $\square r_0 = \max_t |r(t)|$
- Design peak values of forces and displacements
- Spectral analysis: direct determination of peak values of forces and displacements
- Peak response r_{no} of the contribution r_n(t) in the n-th mode to the total response r(t)

$$r_{n0} = r_n^{st} A_n$$

A_n - spectral pseudo-acceleration

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Modal contrib. and total time-history response



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Methods for combination of peak modal response

Absolute sum

$$r_0 = \sum_{n=1}^{N} |r_{n0}|$$

suitable for structures with closely spaced natural modes of vibration

Square Root of Sum of Squares (SRSS):

$$r_0 = \sqrt{\sum_{n=1}^{N} r_{n0}^2}$$

suitable for structures with distinct modes of vibration

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Methods for combination of peak modal response

Complete quadratic combination (CQC):



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Dynamics of structures

Spectral analysis: summary

- Define structural properties
 - mass [m] and stiffness [k] matrices
 - critical damping ratio ξ_n
- Determine $\omega_n (T_n = 2\pi/\omega_n)$ and $\{\phi\}_n$
- Response in *n*-th mode:
 - T_n and $\xi_n \Rightarrow$ pseudo-acceleration A_n from the response spectrum
 - equivalent static forces $\{f\}_n = \{s\}_n A_n$
 - compute response quantity r_n from forces $\{f\}_n$, for each response quantity
- Combine modal contributions r_n to obtain total response using SRSS or CQC combination methods
- Note: generally it is NOT necessary to consider ALL modes of vibration

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to earthquake engineering •Introduction to earthquake seismology	Dyn	amics of structures
•linear analysis	Spect	tral analysis: summary
•nonlinear analysis		
•Ductility and nonlinear behavior	Define properties of the structure:	[<i>m</i>]
•Design response spectra (UBC & IBC)	- mass matrix [<i>m</i>] and stiffness matrix [<i>k</i>] - critical damping ratio ξ _n	[k] <u> </u> <u> </u>
•Method of analysis – Force based (equivalent lateral force)	Find out natural circular frequencies ω_n (with the corresponding periods $T_n = 2\pi/\omega_n$) and natural modes of vibration $\{\phi\}_n$	$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$
and joint design		$\{\varphi\}_1, T_1 \qquad \{\varphi\}_2, T_2 \qquad \{\varphi\}_3, T_3$

issues



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BC rce nt	Compute the total response r by combining modal contributions r_n (e.g. using the SRSS method)	$M_{A} = \sqrt{M_{A1}^2 + M_{A2}^2 + M_{A3}^2}$
		r